

AQ1 **Additive Manufacturing and**
 AQ2 **Performance of Functional Hydraulic**
 2 **Pump Impellers in Fused Deposition**
 AQ3 **Modeling Technology**

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38 *Would it be possible for an additively manufactured impeller of*
 39 *fused deposition modeling (FDM) technology to have a functional*
 40 *behavior and a similar performance to that of the original impel-*
 41 *ler of a close-coupled centrifugal pump? In this research paper,*
 42 *different tests are conducted to answer this question and to evalu-*
 43 *ate the manufacturing process of FDM functional parts. Three*
 44 *performance experiments with the same centrifugal pump, using*
 45 *an open test rig, are carried out and compared, first using the*
 46 *original impeller provided by the manufacturer of the pump,*
 47 *second using an FDM replication of the original one without*
 48 *post-treatment, and third using a chemically dimethyl ketone post-*

treated FDM replication. The results obtained in the tests demon-
 49 *strate the functional behavior of both additive FDM impellers in*
 50 *comparison with the one fabricated by means of conventional*
 51 *technology (subtractive manufacturing). Additionally, analogous*
 52 *head-flow curves (and also with an improved performance to the*
 53 *one of the original impeller) are obtained. This research paper*
 54 *introduces significant information concerning a low-cost and low-*
 55 *time manufacturing process of additive functional parts. More-*
 56 *over, new results are presented regarding the performance of*
 57 *chemically post-treated FDM parts working in functional applica-*
 58 *tions. FDM impellers of high complexity and quality, which meet*
 59 *performance criteria, can be achieved. [DOI: 10.1115/1.4032089]*
 60 AQ5

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1 Introduction 63

FDM is a layer-by-layer additive manufacturing (AM) technol- 64
 ogy that enables fabrication of physical objects directly from 65
 CAD data using a heated thermoplastic filament extruded through 66
 a nozzle for building parts by incremental material deposition 67
 [1–3]. In contrast to classical methods of manufacturing, these 68 AQ6
 processes are based on the additive principle for the fabrication of 69
 parts, whose main advantages are no limitation with regard to 70
 complex geometries and building time reduction depending on the 71
 geometrical complexity of the model, the requirements of the con- 72
 ventional process and the size of the production batch. At the pres- 73
 ent time, FDM is used not only for concept models and prototypes 74
 but also for final functional parts [3,4]. 75

There is a previous research about the manufacturing and per- 76
 formance of the FDM impellers of pumps. FDM is used with 77
 a time-cost reduction approach in order to redesign the blades of a 78
 turbine [5]. A mixed impeller is tested, with FDM blades and a 79
 metallic body. FDM is evaluated in order to decide if is optimal 80
 for fabricating a prototyping model of a pump impeller [6]. The 81
 mean roughness (Ra) of the FDM model is 12.5 μm and, therefore, 82
 the authors indicate the requirement to polish and coat the model 83
 with some material to seal its porous surfaces. Consequently, and 84
 due to the outstanding disadvantage of this model, the FDM tech- 85
 nology is discarded in the study. And finally, the results in Ref. 86
 [7] have shown that FDM is a viable and feasible method of pro- 87
 ducing impellers to be tested and to help the pump designer to test 88
 low-cost prototypes of new and complex blade geometries. 89

So as to make a contribution beyond other previously published 90
 investigations, in this research paper: 91

- Entire FDM impellers and not only FDM blades will be 92
 studied.
- The performance tests will be extended beyond prototypes 93
 for preliminary testing.
- The roughness of FDM impellers without post-treatment 94
 will be analyzed as a potential problem or limitation regard- 95
 ing its proper performance.
- The performance of chemically post-treated FDM impellers 96
 operating in functional applications will be investigated.

The main purpose of this investigation aims to obtain impellers 97
 with a hydraulic capacity and high performance using FDM and a 98
 subsequent finishing external chemical process. 99

In this study, the performance of three centrifugal pump impel- 100
 lers will be tested and compared: the original one provided with 101
 the centrifugal pump and two additively manufactured replica- 102
 tions of the original, one untreated and one post-treated, in order 103
 to evaluate the effect of a chemical external agent on the strength 104
 of the impeller. The post-treatment process is based on the immer- 105
 sion of the impeller in a dimethyl ketone water solution which dis- 106
 solves the material externally and improves the surface quality of 107
 the part [8]. 108

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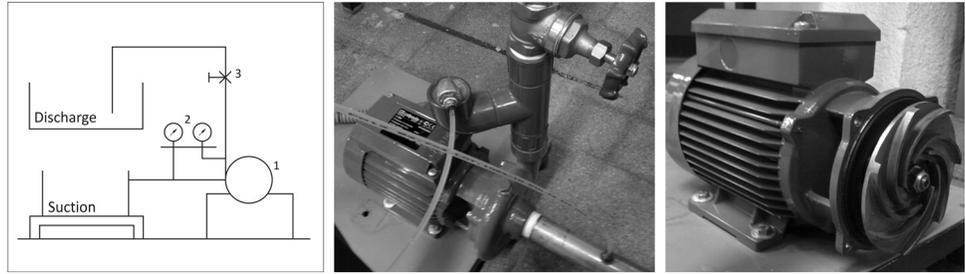


Fig. 1 Test rig: pump (1), pressure gages (2), and valve (3)

109 **2 Materials and Methods**

110 **2.1 The Studied Application.** The performance of the impellers is experimentally investigated in an open test rig (Fig. 1) used to measure the flow rate with seven complete test cycles per impeller, according to ISO 9906 standard—hydraulic performance acceptance tests for rotodynamic pumps. The water level of the suction tank is maintained constant by another external pump.

116 The tested application is a close-coupled horizontal and centrifugal pump with a semi-open impeller. The model reference of the pump “Calpeda C 20 E B-C20/A” is driven by a three-phase AC electric motor.

120 **2.2 The Impeller CAD Model.** The process begins with the conversion of the impeller CAD model into the impeller STL model using SOLID EDGE ST6 software (conversion tolerance of 0.01 mm and surface plane angle of 0.5 deg) (Fig. 2).

124 **2.3 Decision of the AM Technology.** The main AM categories according to ISO 17296-2:2015 are: vat photopolymerization, material jetting, binder jetting, powder bed fusion, material extrusion, directed energy deposition, and sheet lamination. The AM technology of a part should be properly chosen depending on the main aim that is supposed to be achieved: aesthetic, functional, experimental, or visual [9]. The FDM technology (material extrusion category) is selected to produce the impellers owing to its tensile, bending and impact strengths, its heat stability, and its chemical resistance to achieve functional parts [5] (Fig. 2). Moreover, a plastic-based AM technology is chosen over a metal-based AM technology considering that pumped liquids with acid

inclusions are the main cause of corrosion failure of metal impellers [10]. Plastic impellers made of acrylonitrile butadiene styrene (ABS), polyethylene or PVC are used in most installations and have the advantages of being lightweight and corrosion-resistant [11,12]. In addition to this, the FDM technology is also selected due to its cost given that two economical materials associated to FDM can be employed, ABS and polylactic acid (PLA). ABS is finally selected for the reason that in comparison to PLA, ABS has a longer lifespan and a higher strength. PLA is more brittle than ABS and will tend to splinter and break [13].

2.4 The FDM Impellers. The FDM impellers have been fabricated by depositing incremental layers in the XY plane. As far as FDM technology is concerned, the orientation of the impeller on the working tray of the machine is a crucial variable of the process. The impellers are built in the Z orientation on the XY layers as the obtained surface roughness of the flow direction surfaces is of a higher quality than one of the impellers built in the X and Y orientations. Additionally, in this Z orientation the profile of the blades is the optimal one because the geometry of its contour is fabricated in the XY plane.

To this aim, the “Stratasys Dimension SST 768” commercial machine and the Catalyst 4.4 application for interfacing have been employed. In this research, the parameters of the process are: a 0.254 mm layer resolution since it is the most accurate that the machine is able to perform; a solid model interior given that it is the appropriate one for functional parts; and a smart support fill due to it being optimized from the point of view of material used and being the one that interacts less with the fabricated part. Therefore, a postprocess removal is avoided and the results of the research are protected from being affected by this removal. And the parameters preset by the machine manufacturer are: a chamber temperature of about 74 °C and an extrusion nozzle temperature of about 102 °C.

The material, time, and cost comparison between impellers can be summarized as follows:

- FDM impeller without treatment: 28.6 g mass, 30.94 cm³ model material, 18.30 cm³ support material, 3 hrs manufacturing time, and 40€ manufacturing cost.
- FDM impeller with treatment: 28.4 g mass, 30.94 cm³ model material, 18.30 cm³ support material, 3 hrs manufacturing time, and 40€ manufacturing cost.
- Conventional machined metal impeller (unitary production): 251.6 g mass, 2d manufacturing time, and 150€ manufacturing cost.

The FDM cost includes the materials (model, support, and tray), the machine (price, amortization, and maintenance), and the manpower costs of the preparation, manufacturing and post-treatment processes, and this cost has been calculated according to the recommendations in Refs. [14–16]. The difference between the mass of the original and the FDM impellers only influences the overall efficiency of the pump in terms of the mechanical efficiency (rotating mechanical masses and mechanical losses, with

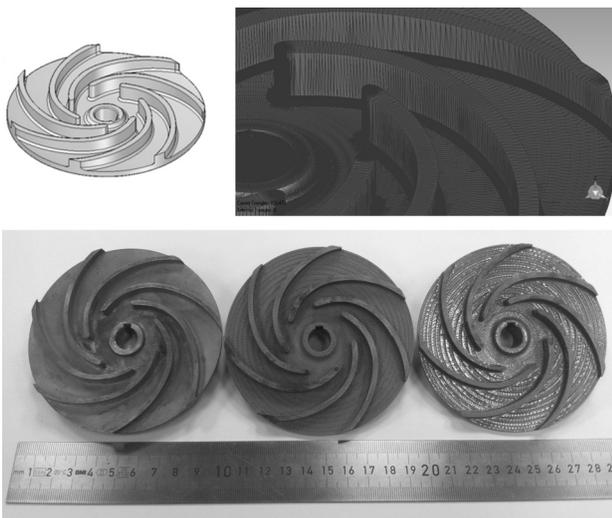


Fig. 2 CAD and STL models, original impeller (Brass P-Cu Zn 40 Pb 2 UNI 5705), untreated and post-treated FDM impellers (mm scale). Stratasys Dimension SST 768. Build platform, X-Y-Z axes definition and built orientation of the impellers.

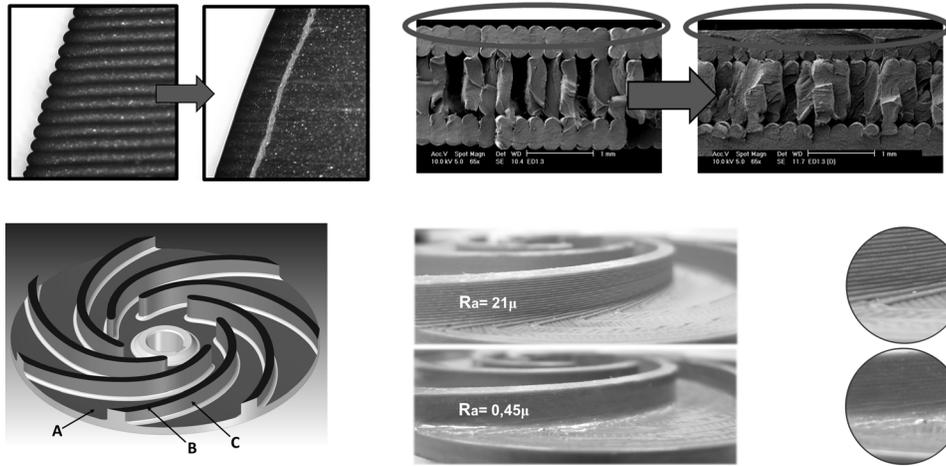


Fig. 3 Effect of the dipping bath in the FDM test specimens of previous investigations of the authors obtained using a scanning electron microscope. Rear shroud (A) and open blades (B top, C side) of the impeller. Ra of the side of the open blade.

185 no consideration given to hydraulic effects). Hence, this differ-
 186 ence is not expected to affect the performance results of the test,
 187 given that once the operating steady-state is reached, the mass of
 188 the impeller does not influence the head-flow curve of the pump
 189 [17–19].

190 **2.5 Post-Treatment of the FDM Impeller.** The surface
 191 roughness controls the magnitude of the hydraulic and friction
 192 losses in the impeller of a pump [20–22]. The first impeller stud-
 193 ied is an untreated one and its mean roughness will be analyzed as
 194 a potential limitation regarding the proper performance of the
 195 pump. The second impeller studied is a post-treated one so as to
 196 improve its surface quality and test an FDM impeller with a simi-
 197 lar roughness to the one of the original impeller of the pump, and
 198 concurrently to analyze the effect of an external and chemical
 199 agent on an FDM functional part and to evaluate its mechanical
 200 behavior taking into consideration the results presented in previ-
 201 ous research papers of post-treated FDM test specimens in
 202 comparison with raw ABS test specimens [23–25]. The chemical
 203 post-treatment process involves a 20-s dipping bath of the impe-
 204 ller using acetone (CH₃)₂CO (dimethyl ketone) which melts the
 205 top layers, enhancing the surface finish of the FDM part (Fig. 3).
 206 The organic solvent containing strong polar groups overrules
 207 polymer–polymer interactions in order to consolidate the polymer
 208 segments of a plastic that exhibits strongly polarizable side
 209 groups.

210 **3 Analysis and Results**

211 The arithmetic average of the absolute values of the roughness
 212 profile ordinates, the mean roughness (Ra), for the rear shroud and
 213 for the open blades of the three impellers used in this research is
 214 measured using the Zeiss SURFCOM 1500 roughness tester
 215 (Fig. 3).

216 The results of the surface measurement can be summarized as
 217 follows:

- Original impeller pump: (A) Ra = 0.67 μm, (B) Ra = 1.6 μm, (C) Ra = 1.5 μm.
- FDM impeller without treatment: (A) Ra = 19.9 μm, (B) Ra = 13.9 μm, (C) Ra = 21 μm.
- FDM impeller with treatment: (A) Ra = 1.1 μm, (B) Ra = 0.7 μm, (C) Ra = 0.45 μm.

221 Afterward, the three head-flow curves of the pump are obtained
 222 (Fig. 4) from the results of the test cycles of the impellers. An

average downward deviation of 0.2% from the reference curve of
 the original is observed in the heads of the untreated FDM impeller
 curve. On the contrary, an average upward deviation of 2.1%
 from the reference curve of the original is observed in the heads
 of the treated FDM impeller curve. These deviations are clearly
 noticeable in the high flow operating range of the curve.

On the one hand, the inherent porosity of the untreated FDM
 impeller adversely affects the performance of the pump owing to
 liquid losses in the inner geometry of the impeller. On the other
 hand, an increase in the volumetric efficiency of the pump using
 the treated FDM impeller is observed. In other words, the liquid
 losses in the pump during the compression process have decreased
 due to the effect of the solvent in the outer geometry of the impe-
 ller, seeing as the geometry of the impeller slightly widens and the
 inner clearances are reduced.

Moreover, both phenomena and their relationship with the origi-
 nal impeller curve are directly connected to the difference
 between the mean roughness of the original impeller and the FDM
 impellers, an improved one in the case of the treated and a wors-
 ened one in the case of the untreated. In order to obtain optimum
 performance in a centrifugal pump, the water passages should be
 as smooth as possible. Despite the fact that the degree of surface

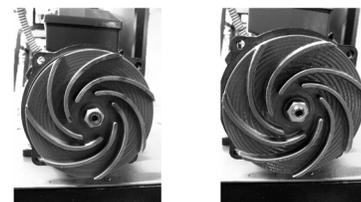
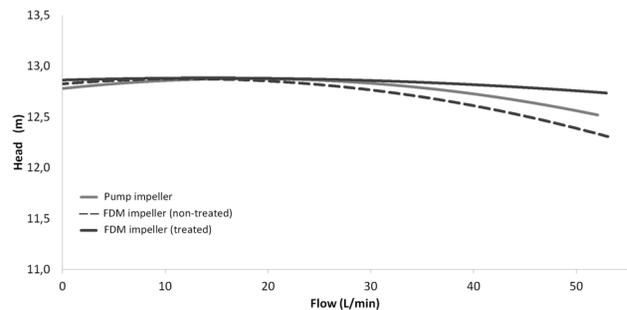


Fig. 4 Head-Flow curves of the three tested impellers. The impeller mounted on the pump after the tests with (a) and without (b) chemical post-treatment.

245 finish has no effect on the general shape of the head curves, the
 246 performance is higher in the case of the smooth impeller and like-
 247 wise more stable in the high flow operating range. The roughness
 248 of the impeller surfaces decreases the pump performance [21].

249 The height, pitch, and density of the elements of roughness as
 250 well as their direction with respect to the flow have an important
 251 impact on the losses [22]. An enhancement in the surface quality
 252 of an impeller achieves a 2–3% improvement of the performance
 253 of the pump [26] and the results in the study of this research paper
 254 are exactly within this range.

255 At the end of this experiment, none of the FDM impellers have
 256 suffered any damage or degradation (Fig. 4). The useful life of
 257 plastic impellers is expected to be at least equal to the one of
 258 metal impellers given the fact that ABS has proper mechanical
 259 properties and if the pumped liquid is chemically neutral to the
 260 plastic of the impeller [27].

261 4 Conclusions and Future Work

262 In conclusion, in this study, AM and FDM have been demon-
 263 strated to be low-cost and low-time processes used to manufacture
 264 impellers that render the performance of the original ones in
 265 standardized head-flow curve tests. The FDM impellers were fab-
 266 ricated in a noticeably reduced time in comparison with that of the
 267 conventional manufacturing technology, with a material and finish-
 268 ing treatment that meet the mechanical and hydraulic require-
 269 ments, and at a competitive cost. These functional impellers could
 270 be used in the approval processes for updating and redesigning
 271 new impeller models to meet the requirements for pump perform-
 272 ance according to international regulations.

273 The main contributions and findings of this research paper are:

- 274 • That an impeller fabricated using AM FDM technology (an
 275 entire FDM impeller and not only FDM blades) has a func-
 276 tional behavior and a similar performance to the original
 277 impeller of the rotodynamic hydraulic pump.
- 278 • That the inherent roughness in the FDM manufacturing pro-
 279 cess of the external surfaces of the impeller is not a limita-
 280 tion in the results of the head-flow curve of the pump.
- 281 • That the improvement of the surface quality of the FDM
 282 impeller using a low-cost chemical post-treatment provides
 an enhancement in the performance of the pump and a
 greater stable behavior in the high flow operating range of
 the pump.

283 Further developments are being conducted by the authors in
 284 addition to this study, so as to achieve advanced and significant
 285 results and conclusions with regard to FDM additive impellers
 286 and their performance. The study that is described in this research
 287 paper is being repeated with impellers of different materials, with
 288 semi-open impellers of different geometries and with closed
 289 impellers, making use of test rigs that consist of not only an FDM
 290 impeller but also an FDM pump body.

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